

1 June 29, 2001
2 Express Mail Label No. ET582225355US PATENT

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11 UNITED STATES PATENT APPLICATION
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24 MACHINE FOR MANUFACTURING A CAPITAL FOR AN ARCHITECTURAL COLUMN
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BACKGROUND OF THE INVENTION

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FIELD OF THE INVENTION

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This invention relates to a machine used to rotate the mold of a capital for an architectural column in the process of manufacturing a capital from elastomers and foams or elastomer-foam combinations. As used throughout this document, the term "elastomer" is meant also to include a "resin"; and the term "foam" includes any liquid that will expand and create a lightweight filler.

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DESCRIPTION OF THE RELATED ART

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The inventor is unaware of any patents relating to a machine used for rotating molds for architectural structures to produce elastomer or elastomer-foam products.

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Traditionally, due to the difficulty in working with elastomers or both elastomers and foam, capitals for architectural columns have not been manufactured from these materials. The process for making such capitals preferably involves rotation, as described in copending United States patent application serial no. 09/862,893, filed on 05/22/2001. And it is further preferable to accomplish such rotation with a machine.

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It is assumed by the inventor that a machine to aid in the process of making capitals for architectural columns from elastomers or both elastomers and foam, has not been constructed due to the difficulty presented in finding and/or utilizing a process by which capitals could be produced using these materials, with or without the machine, resulting in a desirable outcome.

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SUMMARY OF THE INVENTION

2 In the process of making architectural capitals from elastomers and elastomer/foam
3 combinations it is preferred that the mold of the capital be rotated about multiple, preferably two,
4 axes to assure that the elastomers cover the entire inside surface of the mold.

5 Rotation of the mold can be accomplished in any manner but due to the difficulty of
6 rotating the mold by hand it is preferably done by machine.

7 The Machine of the present invention has a shaft that is rotated about multiple axes,
8 preferably substantially orthogonal axes, and most preferably the roll axis and also the yaw axis.

9 The shaft is removably attached to the mold of the capital.

10 Preferably, the shaft extends symmetrically about the point of rotation, or fulcrum for
11 rotation, for the first, preferably the pitch, axis. This permits a mold to be attached at each end of
12 the shaft and thereby balances the load on the shaft.

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BRIEF DESCRIPTION OF THE DRAWINGS

2 Figure 1 depicts a typical setup which allows the capital mold to be attached to the
3 Machine.

Figure 2 depicts one possible machine design that will rotate a mold upon two axes to aid in the process of making capitals from elastomers or both elastomers and foam.

Figure 3 depicts the same Machine design as that of figure 2, but from a different viewpoint.

Figure 4 depicts another possible Machine design that will rotate a mold upon two axes to aid in the process of making capitals from elastomers or both elastomers and foam.

Figure 5 depicts the same Machine design as that of figure 4, but from a different viewpoint.

Figure 6 depicts yet another possible Machine design that will rotate a mold upon two axes to aid in the process of making capitals from elastomers or both elastomers and foam.

Figure 7 depicts the same Machine design as that of figure 6, but from a different viewpoint.

1 DESCRIPTION OF THE PREFERRED EMBODIMENT

2 To a capital mold **1** is attached a first portion of a connector, preferably a pipe
3 attachment, **3** for releasably connecting the mold to a shaft **2** of a Machine. Attachment of the
4 pipe attachment **3** can be by any means that is well known in the art, such as, but not necessarily
5 limited to, screws or bolts and nuts, as depicted in Figure 1.

6 The capital mold **1** is preferably, but not necessarily, placed upon the shaft **2** of the
7 Machine (Figures 2-7) by inserting the shaft **2** into the pipe attachment **3** upon the capital mold **1**.
8 The shaft **2** is inserted until vice grips **4**, of which one jaw **5** has been welded to the pipe
9 attachment **3**, slide past the slip ring **6**. The vice grips **4** are then tightened onto the shaft **2**.

10 The strength of the vice grips **4** alone, clamped onto the shaft **2**, is sufficient to keep the
11 capital mold **1** upon the machine (figures 2-7) during operation, and therefore the slip rings **6** are
12 not fully necessary. However, due to safety precautions, these slip rings **6** are used so that if
13 there is any slippage of the vice grips **4** upon the shaft **2** during operation of the machine (Figures
14 2-7), the vice grips **4** will not be able to slide further down the shaft **2** than the position of the slip
15 rings **6**. This prevents the mold **1** from being dropped. The pipe attachment **3** is, thus, preferably
16 of such a dimension that the outside diameter of the slip rings **6** is less than the inside diameter of
17 the pipe attachment **3**.

18 The preceding is merely one example of a releasable connector for attaching the mold **1**
19 to the shaft **2**. Of course, any releasable connector that is known in the art could be utilized.

20 In order to put less stress upon the machine (Figures 2-7) and its components, it is
21 preferred that the shaft extend substantially symmetrically about the point for which rotation will
22 occur for the pitch axis, *i.e.*, the fulcrum about which rotation will occur in the pitch axis, so that
23 a mold **1** can be attached near both ends of the shaft and that the weight upon both ends of the
24 shaft **2** will be approximately balanced (It is preferred but not necessary that molds **1** of
25 approximately equal weight are placed upon both shafts **2** and that the capitals for these
26 respective molds **1** are made during the same period of time. If this is not desired, another
27 plausible solution would be to attach weights, whose sum is approximately equal to the weight of
28 the mold **1**, on the shaft **2** opposite that of the mold **1**.).

29 Once the mold(s) **1** and/or weights are securely attached to the Machine (Figures 2-7)
30 upon the shafts **2** the Machine (Figures 2-7) is then started. The rotation of the capital mold **1**

1 upon the machine (Figures 2-7) is provided by any number of possible arrangements of
2 machinery which may be composed of, but not limited to: motors, belts, chains, levers, gears,
3 transmissions, and differentials. Figures 2-7 show three examples different Machine designs that
4 can be used. The critical feature is that the Machine rotate the mold 1 about multiple axes,
5 preferably two and preferably substantially orthogonal axes.

6 A description of the components and the functions of these components for the three
7 exemplary Machines, which produce the desired result (i.e., rotation of the mold 1 upon two
8 axes), follows. In each of the drawings for these examples the belts and/or chains (whichever is
9 preferred) which attach one pulley to another have not been drawn in order to allow a clear view
10 of the other components of each machine. It will be assumed that they are there and do exist,
11 though not drawn, for it is by these belts and/or chains which energy is transferred from one
12 point to another in the machine system.

13 Figure 2 and figure 3 show the same Machine 7 at different angles, this Machine 7 being
14 one of the possibilities to achieve rotation upon two axes. In this Machine 7 the pulley 8 of a
15 motor 9 is used to turn the pulley 10 of a differential 11 which in turn rotates another pulley 12
16 of that differential 11. That pulley 12 of the differential 11 then rotates a different pulley 13
17 which is attached and fixed rigidly to a bar 14. This bar 14 then rotates precisely with the
18 rotation of this pulley 13. Upon another part of the bar 14 is also rigidly fixed a lever arm 15.
19 As the bar 14 rotates, the lever arm 15 rotates precisely with it. At another point upon this lever
20 arm 15 is freely attached second lever arm 16 allowing the second lever arm 16 to rotate freely
21 about a pin 17 which attaches it to the first lever arm 15, and upon a plane parallel to the first
22 lever arm 15. At another place on this second lever arm 16 is freely attached a second pin 18
23 which is rigidly attached to a table 19. This table 19 is supported by two bars 20. Through
24 these bars 20, and through the center of the table 19, passes a third pin 21, allowing the table 19
25 to freely rotate about this pin 21. Hence as the pulley 13 attached to the bar 14 is rotated, and in
26 turn rotates both the first lever 15 arm and the second lever arm 16, the table 19 is rotated, or
27 pivoted, about the third pin 21. This is due to the second pin 18 which attaches the second lever
28 arm 16 to the table 19. This angular motion, the table 19 pivoting about the third pin 21, gives
29 the mold 1 attached to the shaft 2 rotation about the pitch axis. To obtain rotation about a second
30 axis a second motor 22 is placed upon the table 19 which rotates the bar ends 2 orthogonally to

1 the pivoting motion of the table **19** about the third pin **21**, *i.e.*, about the roll axis. The angular
2 velocity of the rotation of the ends of the shaft **2** is controlled and can be changed by a
3 transmission **23** mounted also upon the table **19**. The angular velocity of the rotation of the table
4 **19** about the third pin **21** is controlled by the speed of the first motor **9**. This is desirable since
5 adjustments of the velocities of rotation about the axis is preferred.

6 In Figure 4 and Figure 5 a second Machine **24** design possibility is presented. Here, the
7 pulley **8** of a motor **9** turns the pulley **10** of a differential **11** which in turn rotates another pulley
8 **12** of the differential **11**. The rotation of the second pulley **12** of the differential **11** rotates a
9 lever arm **15**. This lever arm **15** in turn rotates a second lever arm **16** as it is freely attached to
10 that second lever arm **16** by a pin **17** (as previously described with the first machine **7** design
11 possibility above). As it is attached to a table **19** by a second pin **18** (as previously described
12 with the first machine **7** design possibility above) the movement of the second lever arm **16**
13 causes the table **19** to pivot about a third pin **21**. This third pin **21** passes through both two
14 supporting bars **20** and the table **19** (as previously described with the first machine **7** design
15 possibility above). This allows for rotation about the pitch axis. For the rotation about a second
16 axis, the pulley **25** of a second motor **26**, which can, but not necessarily is, fixed on top of a
17 differential **27** which is fixed to the table **19**, turns the pulley **28** of the differential **27**. The
18 differential **27** then rotates the shaft **2** upon which are the molds **1** in the roll axis. The velocities
19 of the rotations about the different axes are controlled by the speed of the first motor **9** and the
20 second motor **26**, respectively. The speeds of the first motor **9** and the second motor **26** can be
21 controlled by a control box **29**.

22 Figure 6 and Figure 7 represent yet a third possibility of a machine **30** used to rotate the
23 capital molds **1** about two axes. In these figures the pulley **8** of a motor **9** turns the pulley **10** of a
24 differential **11** which in turn rotates another pulley **12** of the differential **11**. The rotation of the
25 second pulley **12** of the differential **11** rotates a lever arm **15**. This lever arm **15** in turn rotates a
26 second lever arm **16** since it is freely attached to that second lever arm **16** by a pin **17** (as
27 previously described with the first machine **7** design possibility above). Since it is attached to a
28 table **19** by a second pin **18** (as previously described with the first machine **7** design possibility
29 above), the movement of the second lever arm **16** causes the table **19** to pivot about a third pin
30 **21**. This third pin **21** passes through both two supporting bars **20** and the table **19** (as previously

described with the first machine 7 design possibility above). This allows for rotation about the pitch axis. For rotation about a second axis the pulley 31 of a second motor 32 mounted preferably, but not necessarily, to the underneath of the table 19 rotates a pulley 33 through which passes the shaft 2. The rotation of this pulley 33 rotates the shaft 2 in the roll axis. The velocities of the rotations about the different axes are controlled by the speed of the first motor 9 and the second motor 32, respectively. The speeds of the first motor 9 and the second motor 32 can be controlled by a control box 29.

Rotation about a third substantially orthogonal (to both the first axis and the second axis) axis, the yaw axis in the preceding examples, could be accomplish simply by attaching a motor that provides rotary motion to the supporting bars 20.

The pitch, roll, and yaw axes are used herein to denote the same axes as they signify in the case of airplanes.

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